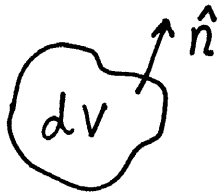


# Lect 12 Stress - Strain - Elasticity of Solids (I)

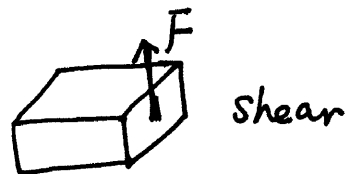
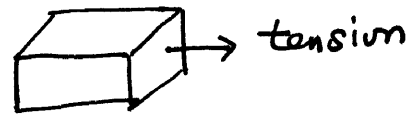
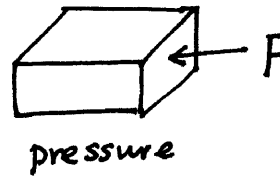
§1 volume forces :



say gravity  $\vec{F} = \rho \vec{g} dV$

electro-static  $\vec{F} = \rho \vec{E} dV$

② surface force



The difference between solids & liquids :

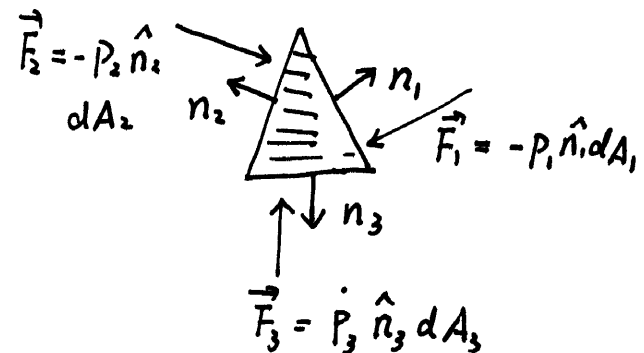
liquids have no shear modulus.

⇒ Isotropic pressure of liquids

Let us choose two surfaces  $S_1, S_2$  normal to arbitrary unit vectors  $\hat{n}_1, \hat{n}_2$ .

Find the 3rd surface  $S_3$  with  $\hat{n}_3$  ⇒ form an isosceles prism

(There're two ends of the prism, but they are normal to the plane). Consider to in-plane force components.



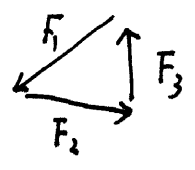
$$\vec{F}_1 + \vec{F}_2 + \vec{F}_3 + \vec{F}_{vol} = m\vec{a}$$

$$\underbrace{\vec{F}_1 + \vec{F}_2 + \vec{F}_3}_{\text{Surface forces}} = \underbrace{m\vec{a} - \vec{F}_{vol}}_{\text{Volume forces}}$$

Let's do scale transformation; shrink the size by a factor  $\lambda$

$$\Rightarrow \lambda^2 (\vec{F}_1 + \vec{F}_2 + \vec{F}_3) = \lambda^3 (m\vec{a} - \vec{F}_{vol}), \text{ as } \lambda \rightarrow 0$$

$\Rightarrow \vec{F}_1 + \vec{F}_2 + \vec{F}_3 = 0$ . Because  $F_{1,2}$  form the same angle with  $F_3$ ,



$\Rightarrow F_1 = F_2 \Rightarrow$  iso-pressure.  
isotropic

direct consequences of no-shear modulus.

## §2. Elastic moduli

Stress: any surface force F proportional the area A

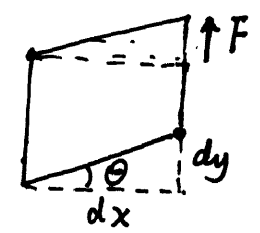
say pressure  $p = \frac{F}{A}$ ,  $\frac{\text{tension}}{A}$ ,  $\frac{\text{shearing force}}{A}$ .

Strain: the change the volume, length, ... (fractional deformation)

$\frac{dV}{V}$  (static fluid),

$\frac{dl}{l}$  (a wire in tension)

$\frac{dy}{dx}$  (for a shear)



$$\text{Stress} = (\text{Young's modulus}) \times \text{Strain} \quad \frac{dF}{A} = YM \frac{dl}{l}$$

$$\text{Stress} = (\text{bulk modulus}) \times \text{Strain} \Rightarrow dp = -BM \frac{dv}{V}$$

$$\text{stress} = (\text{shear modulus}) \times \text{strain} \rightarrow \frac{F}{A} = SM \frac{dy}{dx} \quad \text{etc.}$$

§ The stress-tensor

define an orientational area  $d\vec{A} = \hat{n} dA$  

\* The surface force acting on the area  $d\vec{A}$  is denoted

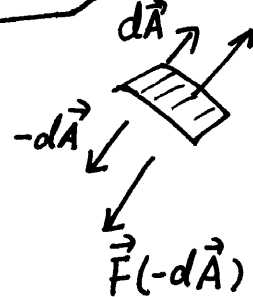
as  $\vec{F}(d\vec{A})$ , we will prove  $\vec{F}(d\vec{A})$  is a linear function of  $d\vec{A}$ , i.e.  $\vec{F}(\lambda_1 d\vec{A}_1 + \lambda_2 d\vec{A}_2) = \lambda_1 \vec{F}(d\vec{A}_1) + \lambda_2 \vec{F}(d\vec{A}_2)$ .

First, as long as for small  $dA$ , we have

$$\vec{F}(\lambda d\vec{A}) = \lambda \vec{F}(d\vec{A}), \quad \text{and} \quad \vec{F}(-d\vec{A}) = -\vec{F}(d\vec{A})$$

$$\vec{F}(d\vec{A}) + \vec{F}(-d\vec{A}) = 0$$

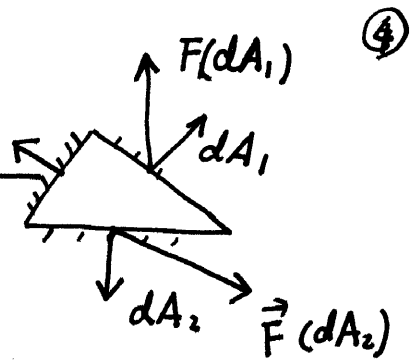
Newton's third law.



Second: Consider two area elements  $dA_1$  &  $dA_2$ ,

find a third area element  $d\vec{A}_3 = -(d\vec{A}_1 + d\vec{A}_2)$

$$\Rightarrow \vec{F}(dA_1) + \vec{F}(dA_2) + \vec{F}(dA_3) = \underbrace{ma}_{\text{Volume force}} \vec{F}_{\text{volume}} + \vec{F}(dA_3)$$



as surface size  $\rightarrow 0$ , right hand side  $\rightarrow 0$

$$\Rightarrow \vec{F}(dA_1) + \vec{F}(dA_2) + \vec{F}(d\vec{A}_3) = 0$$

$$\text{i.e. } \vec{F}(d\vec{A}_1 + d\vec{A}_2) = \vec{F}(-d\vec{A}_3) = -\vec{F}(d\vec{A}_3) = \vec{F}(d\vec{A}_1) + \vec{F}(d\vec{A}_2)$$

Combine with  $F(\lambda dA) = \lambda F(dA)$

$$\Rightarrow \boxed{\vec{F}(\lambda_1 d\vec{A}_1 + \lambda_2 d\vec{A}_2) = \lambda_1 \vec{F}(d\vec{A}_1) + \lambda_2 \vec{F}(d\vec{A}_2)}$$

$\vec{F}$  direction is not necessary along  $d\vec{A}$  if not in liquid.

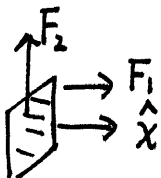
define

$$\vec{F}_i(d\vec{A}) = \sum_{j=1}^3 \sigma_{ij} dA_j \quad (i, j = \hat{x}, \hat{y}, \hat{z})$$

↑

3x3 matrix: tensor

say choose an area element along  $\hat{x}$



$$F_1 = \sigma_{11} dA$$

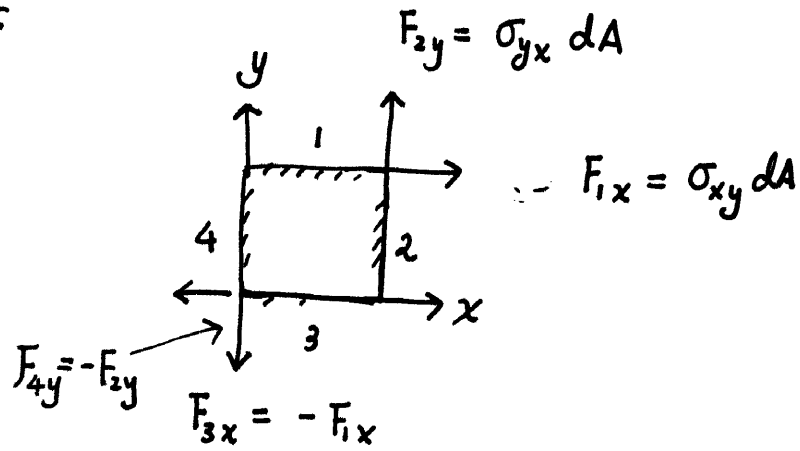
$$F_2 = \sigma_{21} dA$$

§ Stress-tensor is symmetric

the torque  $F_{2y} \cdot l - F_{1x} \cdot l$

$$= (\sigma_{yx} - \sigma_{xy}) l dA$$

$$= \frac{dL_3}{dt}$$



rescale by a factor  $\lambda$  : LHS gains  $\lambda \cdot \lambda^2 = \lambda^3$

RHS gains  $\lambda, \lambda^3 = \lambda^4$

$$\Rightarrow \boxed{\sigma_{yx} = \sigma_{xy}}$$

$\sigma_{ij}$  only has 6-independent elements.

Example: a static fluid :

$$\vec{F} \text{ is along } d\vec{A} \Rightarrow \vec{F}(d\vec{A}) = -p d\vec{A}$$

$$\Rightarrow \Sigma = -p I$$

Generally force does not need to be parallel to surface.

$$\vec{F} = \sum \vec{A}$$