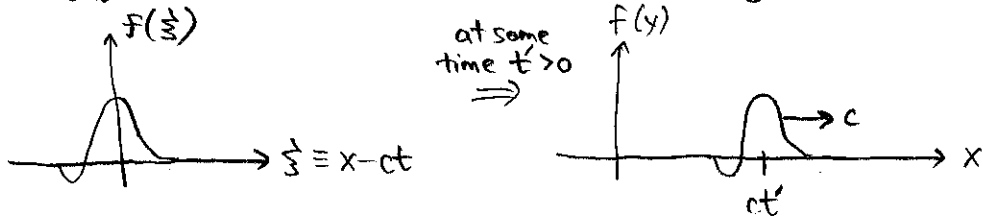


16.12.1

We start w/ some function $f(\xi)$ localized around the origin:

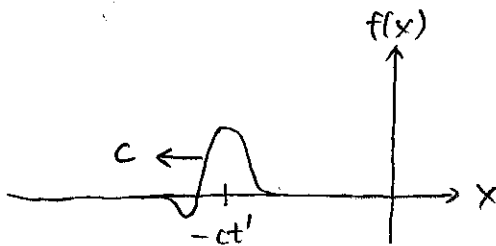


Ostensibly, $f(x-ct)$ is a right-moving wave of speed c .

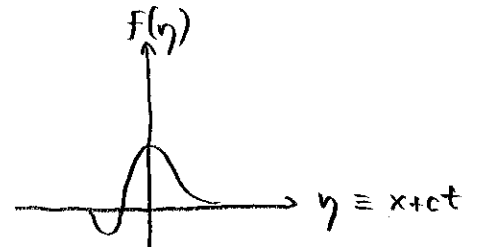
a) We obtain the wave given by the function $f(x+ct)$ by performing $\equiv \eta$

the time inversion: $t \rightarrow -t$

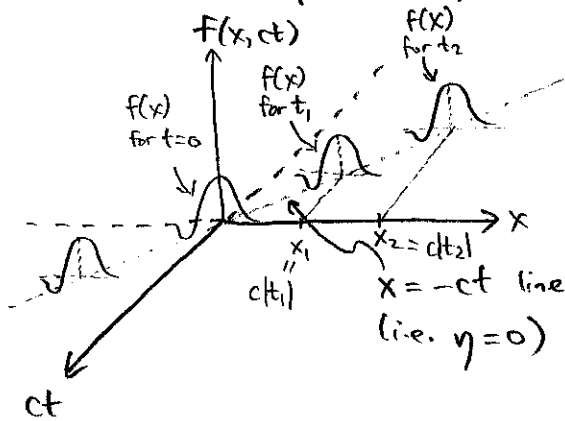
$t' \rightarrow -t'$
 \Rightarrow
 at some time $t' > 0$



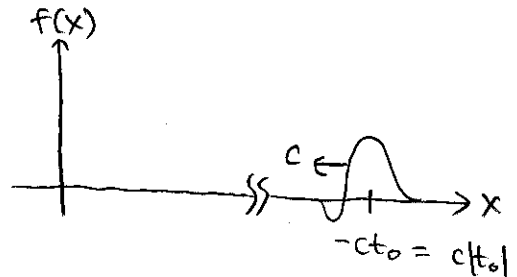
now we have \Rightarrow



Historically, we can plot $f(x, ct)$:



If we want wave given by $f(x+ct)$ for large negative time t_0 , just take $f(x)$ for t_0 :



Thus, $f(x+ct_0)$ is a wave for t_0 to the right (at $x = ct_0$) traveling at speed c to the left.

[6.12]

$$u(x,t)$$

b) We wish to solve for the motion of the wave previously discussed now along a semi-infinite string satisfying the boundary condition:

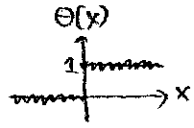
$$u(0,t) = 0 \text{ for all } t; \text{ Obviously, an original } f(x+ct) \text{ does not satisfy this B.C.}$$

We use the method of images to construct the unique solution to the wave equation:

$$\frac{\partial^2 u}{\partial t^2} = c^2 \frac{\partial^2 u}{\partial x^2} \quad \text{or,} \quad \frac{\partial}{\partial \xi} \cdot \frac{\partial u}{\partial \eta} = 0 \quad \text{in terms of coordinates}$$
$$\begin{cases} \eta = x+ct \\ \xi = x-ct \end{cases}$$

$$\text{We are given the function } u(x,t) = f(\underbrace{x+ct}_{\eta}) - f(\underbrace{-x+ct}_{-\xi}).$$

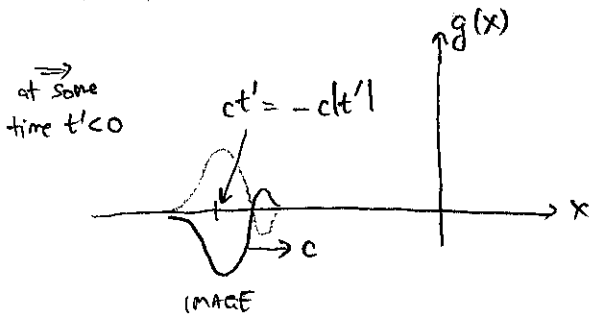
Since the wave must reside on the semi-infinite string, let us include the Heaviside step function as a factor:

$$\Rightarrow u(x,t) = \Theta(x) [f(x+ct) - f(-x+ct)], \text{ where } \Theta(x) = \begin{cases} 1, & x \geq 0 \\ 0, & x < 0 \end{cases}$$


The second term inside is obtained by two successive inversions of the first term:

$$\begin{cases} x \rightarrow -x, & \text{reflection of } f(x) \text{ about } x=0 \\ f \rightarrow -f, & \text{" " " " } f=0 \end{cases}$$

Thus, we have for $-f(-x+ct) \equiv g(x,t)$



thus, $-f(-x+ct)$ is a right-moving wave of speed c . It is ostensibly the twice-reflected "image" of the wave $f(x+ct)$ of part (a). Since we are given that $f(\xi) = f(x-ct)$ is a solution to the wave equation (ignoring BC for now), so is $f(-\eta) = f(-x+ct)$. Thus, $g(x,t) = -f(-x+ct)$ is also a solution.

16.12

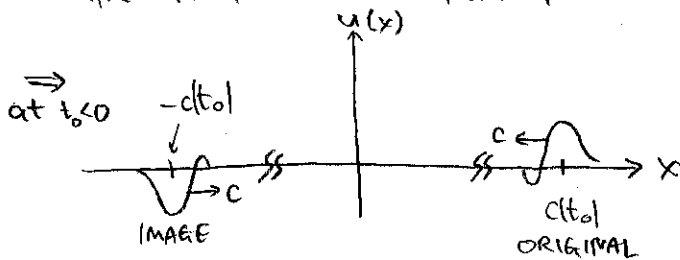
b) By linearity of the wave equation, we have the superposition:

$$\Rightarrow u(x,t) = \Theta(x) [f(x+ct) - f(-x+ct)]$$

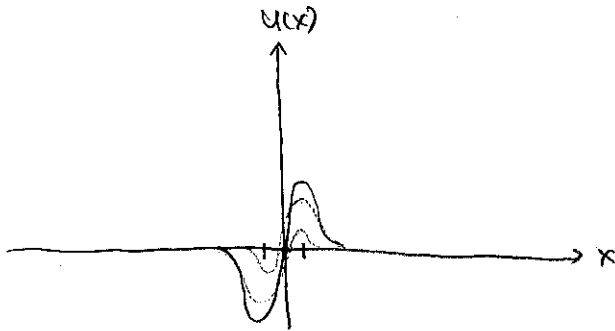
which uniquely solves the wave equation for the BC: $u(x=0,t) = 0$, all t

$$\Rightarrow \boxed{u(0,t) = f(ct) - f(ct) = 0} . \text{ We only need to specify some initial condition.}$$

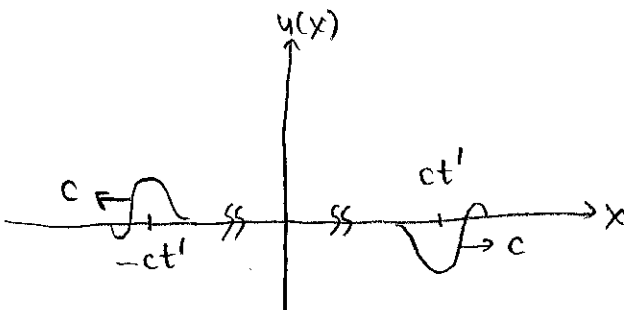
c) The motion on the semi-infinite string can be described as follows for the initial condition from part (a): $u(x,t_0) = f(x+ct_0)$



Ignoring the image term now suppressed by the step function for $t_0 < 0$, the wave travels leftward from far right of the origin.



For t near zero, the original wave starts to interfere w/ the image wave. Notice that the boundary condition $u(0,t) = 0$ is always satisfied.



Eventually for $t' \gg 0$, the image wave is the solution and the original wave is suppressed. (Upon reflection at the origin, the wave is inverted and travels in the opposite direction at speed c)

16.21

We are given the stress tensor $\overset{\leftrightarrow}{\Sigma}$ (a real symmetric matrix) at a pt. P of the continuous medium. By solving the eigenvalue problem $\det(\overset{\leftrightarrow}{\Sigma} - \lambda \overset{\leftrightarrow}{\mathbb{I}}) = 0$, we can find the principal stress axes at P (as we did for the moment of inertia tensor, $\overset{\leftrightarrow}{I}$) and using this basis, write the stress tensor in diagonal form:

$$\Rightarrow \overset{\leftrightarrow}{\Sigma} = \begin{pmatrix} \sigma_{11} & 0 & 0 \\ 0 & \sigma_{22} & 0 \\ 0 & 0 & \sigma_{33} \end{pmatrix}, \text{ where we have rotated the original Cartesian coordinate axes to coincide with the principal stress axes whose directions are now } \hat{x}, \hat{y}, \hat{z}$$

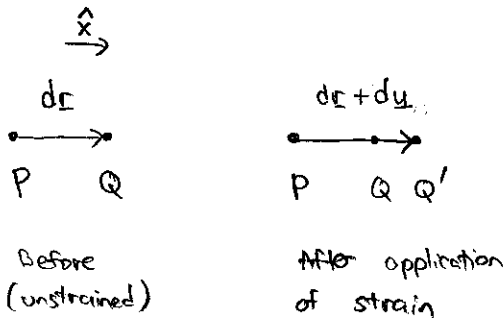
If we consider a surface force on a small element of area dA normal to the x -axis of this basis:

$$\Rightarrow d\mathbf{A} = \begin{pmatrix} dA \\ 0 \\ 0 \end{pmatrix}; \quad \mathbf{F} = \overset{\leftrightarrow}{\Sigma} d\mathbf{A} = dA \begin{pmatrix} \sigma_{11} \\ 0 \\ 0 \end{pmatrix} = \sigma_{11} dA \hat{x}$$

We have found that the surface force on the surface normal to \hat{x} is also normal to the surface (i.e. $\mathbf{F} \parallel d\mathbf{A}$, or $\mathbf{F} \parallel \hat{x}$). We can repeat the exact same argument for the \hat{y} and \hat{z} -directions.

\therefore The surface force on any surface normal to one of the principal stress axis directions is exactly normal to the surface.

16.25



We are given the strain tensor \overleftrightarrow{E} at a pt. P in the continuous solid. Initially, the pt. Q is a distance dr away from pt. P. Since we assume that applied strain leaves P fixed and neighborhood of P unrotated, the displacement dy of Q is in the same direction as dr (chosen to be \hat{x} -direction).

$$dr = \begin{pmatrix} dx \\ 0 \\ 0 \end{pmatrix}$$

a) Generally, we obtain the displacement from:

$$dy = \overleftrightarrow{D} dr \quad (16.75)$$

where $\overleftrightarrow{D} \equiv$ derivatives tensor, which can be decomposed into an asymmetric (\overleftrightarrow{A}) and symmetric (\overleftrightarrow{E}) part:

$$\overleftrightarrow{D} = \overleftrightarrow{A} + \overleftrightarrow{E} \quad (16.80)$$

Since the asymmetric part \overleftrightarrow{A} represents rigid rotations, it vanishes for this problem. not same as shearing!

$$\text{Hence, } dy = \overleftrightarrow{E} dr = \begin{pmatrix} \epsilon_{11} & \epsilon_{12} & \epsilon_{13} \\ \epsilon_{21} & \epsilon_{22} & \epsilon_{23} \\ \epsilon_{31} & \epsilon_{32} & \epsilon_{33} \end{pmatrix} \begin{pmatrix} dx \\ 0 \\ 0 \end{pmatrix} = \begin{pmatrix} \epsilon_{11} dx \\ \epsilon_{21} dx \\ \epsilon_{31} dx \end{pmatrix}$$

But since we are given that pts. in the neighborhood of P are left unrotated,

$dy \parallel \hat{x}$ must hold, thus $\epsilon_{21} = \epsilon_{31} = 0$ (no shearing strains) and we have:

$$dy = \epsilon_{11} dx \hat{x}$$

Thus, $dr + dy = (1 + \epsilon_{11}) dx \hat{x}$ and so the x -axis near P is stretched by a factor of $\boxed{1 + \epsilon_{11}}$

16.25

b) We perform the exact same analysis in part (a) for the \hat{y} and \hat{z} -directions to obtain:

$$\begin{cases} dx + du_x = (1 + \epsilon_{11}) dx \hat{x} \\ dy + du_y = (1 + \epsilon_{22}) dy \hat{y} \\ dz + du_z = (1 + \epsilon_{33}) dz \hat{z} \end{cases} \quad \text{where we have relabeled } dx \text{ and } dy \text{ for clarity}$$

Initially, (before application of strain) the small volume around P is given by:

$$V = dx dy dz$$

The new volume after application of strain is given by:

$$V' = (1 + \epsilon_{11}) dx (1 + \epsilon_{22}) dy (1 + \epsilon_{33}) dz$$

$$\approx (1 + \epsilon_{11} + \epsilon_{22} + \epsilon_{33}) V, \quad \text{where we have ignored terms of higher order } \mathcal{O}(\epsilon^2)$$

$$\text{Thus, } \frac{dV}{V} = \frac{V' - V}{V} = \boxed{\epsilon_{11} + \epsilon_{22} + \epsilon_{33} \equiv \text{tr } \overset{\leftrightarrow}{E}}$$

We can further decompose $\overset{\leftrightarrow}{E}$ into:

$$\overset{\leftrightarrow}{E} = e \mathbb{I} + \overset{\leftrightarrow}{E}', \quad \text{where } e \equiv \frac{1}{3} \text{tr } \overset{\leftrightarrow}{E} = \frac{1}{3} (\epsilon_{11} + \epsilon_{22} + \epsilon_{33}), \text{ the average stretch of the strain tensor, } \overset{\leftrightarrow}{E}$$

so that the first term represents a pure dilatation (i.e. spherical stretching in which each of the stretching directions are stretched by the same amount e) and the second term represents corrections about the pure dilatation.

Generally, $\overset{\leftrightarrow}{E}'$ contains both stretching (diagonal elements) and shearing (off-diagonal) elements but for this problem, off-diagonal elements vanish due to the non-rotation condition.

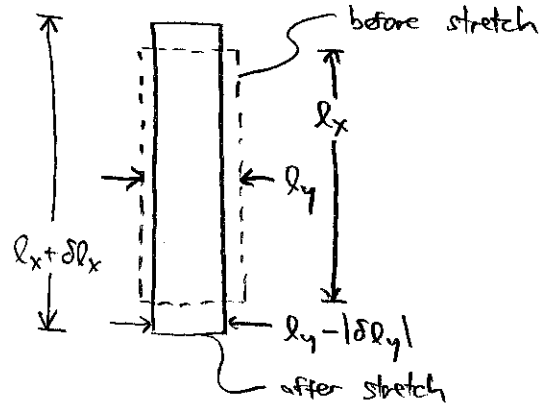
16.28

By definition, Poisson's ratio, $\nu = \frac{\text{transverse fractional contraction}}{\text{longitudinal fractional stretch}} = \frac{-\delta l_y / l_y}{\delta l_x / l_x} \geq 0$

a) for the wire pulled taut in the x-direction:

the ratios we seek are picked out of the strain tensor $\overleftrightarrow{\epsilon}$:

$$\Rightarrow \overleftrightarrow{\epsilon} = \begin{bmatrix} \epsilon_{11} & \epsilon_{12} & \epsilon_{13} \\ \epsilon_{21} & \epsilon_{22} & \epsilon_{23} \\ \epsilon_{31} & \epsilon_{32} & \epsilon_{33} \end{bmatrix}$$



where $\epsilon_{11} = \frac{\delta l_x}{l_x}$ and $\epsilon_{22} = \frac{\delta l_y}{l_y}$

Hence, $\nu = -\frac{\epsilon_{22}}{\epsilon_{11}}$

b) Let us find all elements of the strain tensor, $\overleftrightarrow{\epsilon}$.

From (16.95):

$$\overleftrightarrow{\epsilon} = \frac{1}{3\alpha\beta} \left[3\alpha \overleftrightarrow{\Sigma} - (\alpha - \beta) (\text{tr} \overleftrightarrow{\Sigma}) \mathbb{1} \right]$$

where as given in problem 16.27 we have

$$\overleftrightarrow{\Sigma} = \begin{bmatrix} \sigma_{11} & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}, \text{ due to application of a pure tension along the x-axis. Hence, } \text{tr} \overleftrightarrow{\Sigma} = \sigma_{11}$$

$$\Rightarrow \overleftrightarrow{\epsilon} = \frac{\sigma_{11}}{3\alpha\beta} \begin{bmatrix} 2\alpha + \beta & 0 & 0 \\ 0 & \beta - \alpha & 0 \\ 0 & 0 & \beta - \alpha \end{bmatrix} = \begin{bmatrix} \epsilon_{11} & \epsilon_{12} & \epsilon_{13} \\ \epsilon_{21} & \epsilon_{22} & \epsilon_{23} \\ \epsilon_{31} & \epsilon_{32} & \epsilon_{33} \end{bmatrix}$$

$$\Rightarrow \nu = \frac{\alpha - \beta}{2\alpha + \beta} = \frac{3BM - 2SM}{6BM + 2SM}, \text{ where we have used the definitions of the bulk and shear moduli:}$$

$$\alpha = 3BM \quad (16.97)$$

$$\beta = 2SM \quad (16.99)$$

16.28

c) Poisson's ratio for 5 materials:

⇒ Material	BM	SM	ν
Iron	90	40	.31
Steel	140	80	.26
Sandstone	17	6	.34
Perovskite	270	150	.27
Water	2.2	0	.50

As we can see, for the limit $BM \gg SM$,

$$\Rightarrow \nu = \frac{3BM - 2SM}{6BM + 2SM} \approx \frac{1}{2}, \text{ which matches what we obtained for water.}$$